

1 **APPARATUS FOR GENERATING POWER FROM**
2 **PASSING VEHICULAR TRAFFIC**

3
4 **Background of the Invention**

5 1. Field of the Invention

6 The present invention comprises an apparatus for placement on a roadway to
7 generate power from the weight of passing vehicular traffic.

8
9 2. Description of the Related Art

10 Ever increasing demand for electric power requires that new sources of energy be
11 located and utilized. One untapped resource exists in the form of potential energy stored
12 by the millions of vehicles which travel the world's highways every day. Each moving
13 vehicle has a weight which generally ranges from 2,000 pounds for a small passenger car
14 to 80,000 pounds for a large tractor-trailer truck. If even a small percentage of the energy
15 represented by this vast amount of moving weight could be harnessed and converted to
16 useable form, then the resulting power output would be tremendous.

17 Prior attempts to harness the potential energy of moving vehicular traffic include
18 Roche, et al., U.S. Patent No. 4,212,598, which discloses placing a plurality of pneumatic
19 cylinders or air bladders under respective pivoting actuating panels in a roadway or
20 sidewalk. As vehicle or foot traffic passes over the actuating panels, air is forced out of
21 the cylinders or bladders and circulated to a turbine which is rotated by the flow of air.
22 The turbine is connected to a generator which produces electricity.

1 McGee, U.S. Patent No. 4,614,875, discloses placing a plurality of small rotor and
2 stator type electric generators under a roadway. Each generator is connected to a vertical
3 spindle which is depressed when a vehicle passes thereover. The spindle has screw
4 threads which rotate the rotor of the generator when the spindle is depressed, thereby
5 generating electricity.

6 Galich, U.S. Patent No. 6,172,426, discloses an energy producing platform having
7 a fluid filled bladder positioned therebeneath such that the fluid is compressed by vehicles
8 passing over the platform and forced out of the bladder. The fluid flowing from the
9 bladder is circulated to an accumulator which releases the fluid when a preset pressure is
10 reached. When released from the accumulator, the fluid flows to a generator where the
11 flow is used to produce electricity. In an alternative embodiment of the invention, a
12 moveably mounted platform has a lever arm connected to it such that downward
13 movement of the platform caused by a vehicle passing thereover causes the lever arm to
14 pivot. A hydraulic cylinder is connected to an outboard end of the lever arm such that
15 fluid is forced from the cylinder upon actuation of the lever arm. Again, the fluid is
16 channeled through an accumulator to a generator to produce electricity.

1 Fig. 6 is an enlarged plan view of a telescoping union fitting which is used in the
2 apparatus.

3 Fig. 7 is a cross-sectional view of the union fitting taken generally along line 7-7 in
4 Fig. 6.

5
6 **Detailed Description of the Preferred Embodiment**

7 As required, detailed embodiments of the present invention are disclosed herein;
8 however, it is to be understood that the disclosed embodiments are merely exemplary of
9 the invention, which may be embodied in various forms. Therefore, specific structural and
10 functional details disclosed herein are not to be interpreted as limiting, but merely as a
11 basis for the claims and as a representative basis for teaching one skilled in the art to
12 variously employ the present invention in virtually any appropriately detailed structure.
13 The drawings constitute a part of this specification and include exemplary embodiments of
14 the present invention and illustrate various objects and features thereof.

15 Certain terminology will be used in the following description for convenience in
16 reference only and will not be limiting. For example, the words “upwardly,”
17 “downwardly,” “rightwardly,” and “leftwardly” will refer to directions in the drawings to
18 which reference is made. The words “inwardly” and “outwardly” will refer to directions
19 toward and away from, respectively, the geometric center of the embodiment being
20 described and designated parts thereof. Said terminology will include the words
21 specifically mentioned, derivatives thereof and words of a similar import.

1 Referring to the drawings in more detail and in particular to Figs. 1 and 2, the
2 reference number 1 generally designates a power generation apparatus embodying the
3 present invention. The apparatus 1 includes a mat 3 sized and shaped for placement on a
4 roadway to cover at least one traffic lane thereof. Positioned beneath the mat 3 are a
5 plurality of transverse parallel supports 5 between which are mounted a plurality of
6 hydraulic cylinders 7. Each cylinder 7 includes a piston 9 having an upwardly convex
7 plunger cap 10 which extends above the surface of the mat 3 so as to be engageable by the
8 wheels of passing motor vehicles. When engaged by a vehicle, the cylinders 7 act as
9 pumps to create pressurized flow in a hydraulic fluid. As seen in Fig. 5, the flow created
10 by the cylinders 7 is channeled through a hydraulic circuit 11 to spin a turbine 12 which is
11 mechanically connected to an electric generator 13.

12 Referring again to Fig. 1, the cylinders 7 need not cover the entire traffic lane, but
13 can instead be concentrated into a pair of tracks 14 which are spaced apart to correspond
14 to the wheel tracks of the majority of vehicles. Each of the tracks 14 may include several
15 rows 15 of the cylinders 7 (two rows 15 per track 14 are shown in Fig. 1). An open space
16 17 is left between the tracks 14 which allows for drivers of motorcycles and other light
17 vehicles to avoid driving over the pistons 9 of the cylinders 7 if so desired.

18 Referring to Figs. 3 and 4, each cylinder 7 includes a tubular cylinder body 18
19 forming a cavity 19 in which the respective piston 9 is slidably mounted. The cylinder
20 body 18 has an upper end 20 and a lower end 21. An annular flange 22 extends outwardly

1 from the cylinder body 18 in a location spaced downwardly from the upper end 20 a
2 distance which is substantially equal to the thickness of the mat 3.

3 Each piston 9 includes an upper piston section 23 and a lower piston section 24
4 connected by an intermediate rod 25 such that the piston sections 23 and 24 move in
5 tandem. The upper section 23 has an upper face 26, a lower face 27 and an outer edge
6 28. Similarly, the lower section 24 has an upper face 29, a lower face 30 and an outer
7 edge 31. The plunger cap 10 of each cylinder 7 is connected to the upper face 26 of the
8 respective upper piston section 23 by fasteners such as allen head machine screws 32.
9 Annular grooves 33 formed on the respective outer edges 28 and 31 of the piston sections
10 23 and 24 receive respective sealing rings 34 which seal the gaps between the outer edges
11 28 and 31 and the cylinder body 18.

12 The intermediate rod 25 has an upper threaded end 35 which is received by a
13 threaded axial receiver 36 formed in the lower face 27 of the upper piston section 23 and a
14 lower threaded end 37 which is received by a threaded axial receiver 38 formed in the
15 upper face 29 of the lower piston section 24. The intermediate rod 25 is further secured
16 to each of the piston sections 23 and 24 by a respective flat head machine screw 39 which
17 passes through the respective piston section 23 or 25 and engages a respective threaded
18 receiver 40 in the rod 25.

19 The portion of the cylinder cavity 19 located between the upper piston section 23
20 and the lower piston section 24 comprises a dry cavity 19a which does not receive
21 hydraulic fluid. The portion of the cavity 19 located below the lower piston section 24

1 comprises a fluid cavity 19b which is filled with hydraulic fluid during normal operation of
2 the apparatus 1.

3 The piston 9 is biased upwardly by a compression spring 41 which is positioned
4 within the dry cavity 19a. An upper end of the spring 41 bears against the lower face 27
5 of the upper piston section 23. A lower end of the spring 41 bears against a spacer 42
6 which in turn bears against an upper surface 43 of a shoulder 44 formed on the body 18,
7 the shoulder 44 extending inwardly into the cylinder cavity 19. The shoulder 44 further
8 includes a lower surface 45 which serves as a stop to limit upward movement of the piston
9 9. The lower surface 45 is engaged by the upper face 29 of the lower piston section 24
10 when the piston 9 is at the top of its stroke.

11 The positioning of the spring 41 above the lower piston section 24 allows the
12 spring 41 to be serviced from the top of the apparatus 1 without opening the fluid cavity
13 19b. To access the spring 41, the plunger cap 10 is first removed by unscrewing the
14 screws 32. With the cap 10 removed, the upper flat head machine screw 39 may be
15 removed. The upper piston section 23 can then be rotated off of the intermediate rod 25
16 and removed through the upper end 20 of the cylinder cavity 19, providing access to the
17 spring 41. While the upper piston section 23 is out of the cavity 19, the rings 34 on the
18 upper section 23 may also be replaced.

19 Because each cylinders 7 needs to be serviced at intervals which are dependent
20 upon the number of depressions of the cylinder, a proximity switch (not shown) may be
21 included in each of the cylinders 7 so as to be actuated upon each depression of the piston

1 9. The proximity switch is connected to a counter (not shown) which keeps track of the
2 number of depressions of the piston 9. The counter can then be read to determine when
3 the respective cylinder 7 has undergone a predetermined number of depressions, thereby
4 indicating that the cylinder 7 needs to be serviced.

5 The lower end 21 of the cylinder body 18 is internally threaded to receive an
6 externally threaded upper end 46 of a reducer 47 having an externally threaded inlet/outlet
7 port 48 extending downwardly therefrom. The components of the cylinder 7, including
8 the body 18, piston sections 23 and 24, and reducer 47 are preferably made of marine
9 grade stainless steel.

10 Referring again to Fig. 3, positioned below each cylinder 7 is a respective check
11 valve 71 having a generally rectangular valve body 72 which includes an upper face 73, a
12 first side face 75, and an opposed second side face 77. The upper face 73 has an axially
13 aligned, internally threaded cylinder port 79 formed therethrough having a diameter
14 enabling the port 79 to retainably receive the externally threaded inlet/outlet port 48 of the
15 respective cylinder 7. It should be noted that the single inlet/outlet design of the cylinders
16 7 and check valves 71 allows the cylinders 7 to be directly connected to the check valves
17 71 with a minimum of plumbing. In addition, the design allows the cylinders 7 to be
18 unscrewed out of the apparatus 1 from the top side for ease of maintenance.

19 A supply port 81 is formed in the valve body 72 through the first side face 75. The
20 supply port 81 communicates with a supply cavity 82 formed within the valve body 72.
21 The supply cavity 82, in turn, communicates with the cylinder port 79 through a passage

1 83. The supply cavity 82 receives a supply check valve cartridge (not shown) which
2 allows fluid to flow from the supply port 81 through the passage 83 to the cylinder port
3 79, but prevents flow in the opposite direction.

4 Similarly, a pressure port 84 is formed through the second side face 77 of the valve
5 body 72. The pressure port 84 communicates with a pressure cavity 85 formed within the
6 valve body 72. The pressure cavity 85 also communicates with the cylinder port 79
7 through the passage 83. The pressure cavity 85 receives a pressure check valve cartridge
8 (not shown) which allows fluid to flow from the cylinder port 79 through the passage 83
9 to the pressure port 84, but prevents flow in the opposite direction. Should an unexpected
10 surge in the supply pressure occur, both check valve cartridges will open, allowing fluid to
11 flow directly from the supply port 81 to the pressure port 84 through the passage 83.

12 Each cylinder 7 is positioned between a pair of adjacent supports 5. Each support
13 5 includes an opposed pair of support flanges 87, each of which supports the annular
14 flange 22 of the adjacent cylinder 7. Each support 5 further includes a plurality of
15 opposed pairs of support shelves 88 longitudinally spaced along the support 5 in alignment
16 with the locations of the cylinders 7. Each shelf 88 serves to support the lower end 21 of
17 the cylinder body 18 of the adjacent cylinder 7. The shelves 88 are each reinforced by a
18 gusset 89.

19 Alternate ones of the supports 5, indicated by the reference numeral 5a in Fig.3,
20 serve as pressure manifolds and include a pressure passage 91. Each pressure manifold
21 5a may also include a temperature control passage 92 through which heated fluid may be

1 circulated in cold weather to warm the apparatus 1 and prevent icing of the mat 3. The
2 fluid may be heated by power generated by the apparatus 1. In hot weather, coolant may
3 be circulated through the temperature control passage 92 to cool the apparatus 1. The
4 remaining supports 5, indicated by the reference numeral 5b, serve as supply manifolds
5 and each include a supply passage 93. Each cylinder 7 with its respective check valve 71
6 is thus positioned between a pressure manifold 5a and a supply manifold 5b. Each check
7 valve 71 is oriented with its supply port 81 adjacent a respective supply manifold 5b and
8 its pressure port 84 positioned adjacent a pressure manifold 5a. Threaded receivers 95 are
9 formed in each pressure manifold 5a in alignment with the respective pressure ports 84
10 and in communication with the pressure passage 91. Similarly, threaded receivers 97 are
11 formed in each supply manifold 5b in alignment with the respective supply ports 81 and in
12 communication with the supply passage 93.

13 Each pressure port 84 is connected to the respective threaded receiver 95 by a
14 telescoping union fitting 99. Each fitting 99, as shown in Figs. 6 and 7, includes an outer
15 section 101 with a central bore 102, and an inner section 103 slidably received within the
16 bore 102. The outer section 101 has an externally threaded first end 104 sized to be
17 retainably received by the respective threaded receiver 95, and an opposed second end 105
18 having a hexagonal flange 107 which allows the outer section 101 to be tightened into the
19 receiver 95 with a wrench. An O-ring 108 is positioned proximate the first end 104 to seal
20 against the pressure manifold 5a. The second end 105 further includes a central opening
21 109 sized to receive the inner section 103. An annular thrust surface 111 is formed

1 around the opening 109 on an interior face of the second end 105. The thrust surface 111
2 preferably has an annular groove 113 formed therein for receiving an O-ring 115.

3 The inner section 103 of the fitting 99 has an externally threaded first end 117
4 sized to be retainably received by the respective pressure port 84, and an opposed second
5 end 119 having an outwardly extending flange 121 sized to be received within the central
6 bore 102 of the outer section 101. An O-ring 122 is positioned proximate the first end
7 117 to seal against the check valve body 72. The flange 121 includes an annular thrust
8 surface 123 sized and shaped to bear against the thrust surface 111 of the outer section
9 101. A plurality of wrench flats 125 are formed on an external surface 127 of the inner
10 section 103 such that the section 103 may be tightened into the pressure port 84 using a
11 wrench.

12 When installed, the union fittings 99 are tensioned between the respective check
13 valve 71 and the adjacent pressure manifold 5a. The first end 104 of the outer section 101
14 is screwed into the receiver 95 of the pressure manifold 5a until the O-ring 108 seals
15 against the pressure manifold 5a. Similarly, the first end 117 of the inner section 103 is
16 screwed into the pressure port 84 of the check valve 71 until the O-ring 122 seals against
17 the valve body 72. This draws the thrust surface 111 against the mating thrust surface
18 123, compressing the O-ring 115 therebetween. With the union fittings 99 thus installed,
19 the pressure ports 84 are each placed in communication with the pressure passage 91
20 within the adjacent pressure manifold 5a.

1 Each supply port 81 is connected to the respective threaded receiver 97 in the
2 adjacent supply manifold 5b by a telescoping union fitting 129 which is substantially
3 similar to the fitting 99 previously described, however the threaded ends of the inner and
4 outer sections are sized to be retainably received by the supply port 81 and receiver 97,
5 respectively. The union fittings 129 place the supply ports 81 in fluid communication with
6 the supply passage 93 within the adjacent supply manifold 5b. The unique, telescoping
7 design of the union fittings 99 and 129 allows the cylinders 7 to be spaced in closer
8 proximity to one another than would be possible with standard plumbing fittings.

9 As a passing vehicle engages the plunger cap 10 of a cylinder 7, the piston 9
10 thereof is forced downwardly, building pressure in the fluid cavity 19b. When the pressure
11 in the fluid cavity overcomes the set point of the pressure check valve cartridge, fluid
12 flows out of the fluid cavity 19b, through the check valve 71, and into the pressure
13 passage 91 of the adjacent pressure manifold 5a.

14 As the vehicle passes off of the plunger cap 10 of a cylinder 7, the piston 9 thereof
15 is urged upwardly by the spring 41, creating a vacuum in the fluid cavity 19b. When the
16 vacuum in the fluid cavity overcomes the set point of the supply check valve cartridge
17 (which is set relatively low), the valve opens, allowing fluid to be drawn into the fluid
18 cavity 19b through the check valve 71 from the supply passage 93 in the adjacent supply
19 manifold 5b.

20 Referring to Fig. 5, the hydraulic circuit 11 includes a master supply manifold 131
21 which delivers fluid from a closed and pressurized reservoir 133 to the supply manifolds

1 5b along a section of roadway such that a positive pressure is maintained in the supply
2 manifolds 5b. The pressure manifolds 5a are connected through a master pressure
3 manifold 135 to a master pressure control valve 137 which allows for variation of the
4 operating pressure maintained within the cylinders 7 and pressure manifolds 5a.

5 The amount of power which can be generated by the apparatus 1 is directly related
6 to the operating pressure; thus, the higher the operating pressure, the more power may be
7 produced. The operating pressure at which the apparatus 1 may be operated is, in turn,
8 determined by the weight of the vehicles passing over the cylinders 7. For reasons of
9 safety and comfort to the drivers of the vehicles, it is preferable that the operating pressure
10 in the cylinders 7 be set low enough, via the master pressure control valve 137, that the
11 pistons 9 will move downwardly under the weight of the lightest vehicles by which they
12 are engaged. If it is desired for the operating pressure to be set at a higher level, the
13 apparatus 1 may be placed only in traffic lanes which are dedicated to heavy commercial
14 vehicle traffic. Alternatively, the apparatus 1 may include some sections which are placed
15 in commercial vehicle lanes and are set to operate at higher pressures, as well as other
16 sections which are placed in passenger vehicle lanes and are set to operate at lower
17 pressures.

18 The master pressure control valve 137 may be adapted to be electronically
19 controlled, allowing the operating pressure of the respective section of the apparatus 1 to
20 be varied almost instantaneously. In such applications, the apparatus 1 could also include
21 a sensor 139 which could read the type of vehicle approaching and signal the master

1 pressure control valve 137 to vary the operating pressure to match the particular vehicle.

2 The sensor 139 may comprise a strain gauge or similar weighing mechanism mounted in
3 the road surface, an optical sensor used to determine the size or length of the vehicle, or a
4 receiver which could pick up an identification signal from a transmitter (not shown)
5 mounted on the vehicle.

6 In addition to being dependant on the operating pressure within the cylinders 7,
7 power output of the apparatus 1 is also directly related to the number of pistons 9 which
8 are depressed by each vehicle as it passes over the apparatus 1. In order to multiply the
9 number of pistons 9 which are so actuated, commercial vehicles may be equipped with
10 retractable rollers (not shown) mounted under their trailers which could be extended to
11 engage the pistons 9 of the cylinders 7 when the vehicle is passing over the apparatus 1.
12 This would provide for more cylinder actuations than could be accomplished by the
13 wheels of the vehicle alone.

14 The master pressure control valve 137 also provides a pilot signal which operates a
15 bypass valve 141 and a discharge valve 143 which control flow through an accumulator
16 circuit 145. The circuit 145 includes a bank of accumulators 147 and respective check
17 valves 148. Fluid is provided to the master pressure control valve 137 at adequate
18 pressure for the pilot signal through an orifice 149. When traffic is light and the flow from
19 the cylinders 7 is therefore intermittent, the bypass valve 141 is held closed and fluid flows
20 to the accumulators 147 where it is accumulated to a preset discharge pressure before
21 being released to the turbine 12 through the discharge valve 143, thereby powering the

1 electric generator 13 mechanically coupled to the turbine 12. When traffic is heavier and
2 the flow from the cylinders 7 is relatively constant, a pressure signal is sent from the
3 master pressure control valve 137 which opens the bypass valve 141 and holds the
4 accumulator discharge valve 143 closed. This allows the fluid to flow directly to the
5 turbine 12, bypassing the accumulator circuit 145, and powering the generator 13.

6 The turbine 12 is provided with a turbine protect valve 151 which, if necessary,
7 vents flow to prevent the turbine from over-revving. From the turbine 12, fluid flows back
8 to the reservoir 133 to be re-circulated to the cylinders 7.

9 It is to be understood that while certain forms of the present invention have been
10 illustrated and described herein, it is not to be limited to the specific forms or arrangement
11 of parts described and shown. For example, while the apparatus 1 has been described
12 herein as being hydraulically actuated, it is to be understood that pneumatics could be used
13 instead. It is also to be understood that the turbine 12 and the electric generator 13
14 disclosed herein are only intended to be examples of how fluid flow created by the
15 cylinders 7 can be converted into useful energy and that other power conversion means
16 could be used. For example the turbine 12 could be mechanically coupled to another
17 mechanical device, such as an irrigation pump, instead of an electric generator.

18 Furthermore, the present invention is not to be limited to use in connection with
19 automobile and truck traffic, but is suitable for application to any situation where moving
20 vehicles are passing over a land surface. For example, the apparatus 1 could easily be
21 placed between the rails of a railroad track where the pistons 9 would be engaged by

1 rollers or other structure mounted on the underside of passing trains. For this reason, the
2 term “vehicle” should be interpreted to mean any land vehicle, including trains, and the
3 terms “road” and “roadway” should be interpreted to include railroad tracks and right of
4 ways.

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